

EXPERIMENTAL SIMULATION OF A TENNIS BALL USING WIND TUNNEL

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Abstract. *In this study, tennis balls were analyzed experimentally through the use of wind tunnel with speed ranging from 1m/s to 14m/s, which is a variation in the Reynolds number ($10000 < Re < 60000$). In this context, aerodynamic aspects of the balls were evaluated, including the position of the seam and the degree fuzz, i.e., with and without fuzz. It was possible to analyze the effect of drag on the diameter, in the investigation of the relationship between the drag coefficient (C_D) and the Reynolds number (Re) for new and used balls. Graphics were generated using the Reynolds number and the Drag Coefficient in order to assess the (non)dependency of these parameters. In the measurements performed, the static balls inside the wind tunnel were considered, i.e., without rotation. Therefore, no discussions about the Magnus force are presented. The results obtained, $C_D \approx 3$ to $C_D \approx 0.50$, were consistent for the range of the Reynolds number investigated. High values are expected for the drag coefficient, to the range of Reynolds number examined. The position of the seam, according to Mehta et al. (2008) is negligible to high values of Reynolds, i.e., $Re > 50000$. On the other hand, for low values of Reynold, it can represent a difference of up to about 9% for C_D . The balls without fluff showed the strongest influence of the position of the seam, which characterizes the influence of this parameter. The effect of fuzz seemed to be responsible for about 10% of the total drag for low values of Reynolds number. The diameter variation was analyzed isolately.*

Keywords: *Aerodynamics, tennis ball, wind tunnel.*

1. INTRODUCTION

In sports, many researchers have been studying the aerodynamic characteristics of balls used to the practice of soccer, golf and tennis to allow the improvement of these balls. Specifically in tennis, researches were motivated by the observation that score affected the results of tennis matches, since the ball moves at high speed and the opponent and the audience cannot follow the displacement of the ball. To decrease the speed of score, in 1990, the International Tennis Federation (ITF) decided to carry out researches for achieving a bigger ball, which would directly increase the drag. However, it was noted that the increase in the drag attributed to an increase in the diameter alone would not generate the desired effect (Haake et al., 2000). Hence, the inclusion of studies related to other properties of the balls, such as fuzz, seam and rotation, is desirable, since a change in the rules of the game or in the characteristics of the racket would make the sport unpopular.

The present research considered the ball in a static position in the evaluation section of the wind tunnel, so that the effect of sustenance was considered. Otherwise, it should be inserted a study on the parameters of the Magnus force, observed by the deviation of the trajectory of the ball from a straight path initially expected. This would result in a curve of a lateral deflection in the movement of the ball, besides changes in the sustenance, depending on the rotation of the ball. Thus, the Magnus force results from the change in the displacement points of the boundary layer by the Bernoulli Effect.

Therefore, the main goal of this research is to investigate the behavior of the aerodynamic properties of tennis balls related to the effects of fuzz, seam position in relation to the runoff and effect of the diameter. Considering the knowledge about these properties, new balls were compared with used ones, from the same brand, in order to understand the variation in their behavior throughout its utile life.

2. EXPERIMENTAL PROCEDURE AND EQUIPMENT

The research was based on an investigation of the drag force for low Reynolds values. A wind tunnel of a test section was used to identify the runoff variables of the balls, as shown in Fig. 1, 190mm by 170mm with 200mm long, and a maximal average speed of 14m/s, i.e., 50.4km/h ($Re_{max} = 58000$). Nine values of speed were taken, ranging from each other in about 1.5 m/s (5.4 km/h) in this interval. The analyses were carried out with static ball, within the test section, i.e., without analyzing the effect of rotation, coupled with an instrumented shank, i.e., a scale with 70 mm of height, from the base, suitable for the measurements of the drag and sustenance by the International and English

Systems. The acquisition system provides values of average speed with 0.1 m/s (0.36 km/h) and 0.05 Newtons of resolution, to the speed and drag force, respectively.



Figure 1. Representation of the wind tunnel

The rate of blockage in the case of small sections of tests is relevant. The correction equation used, according to Achenbach (1974):

$$C_{Dcorrected} = C_{Dinitial} \left[\frac{1}{\left(1 + 0,25 \frac{A}{C}\right)^2} \right] \quad (1)$$

In which A is the frontal area of the object and C is the area of test section.

Four tennis balls were used in the research, and their characteristics are shown in Tab. 1. To measure the parameters of diameter and width of the seam, it was considered the simple average number of several measurements performed on a digital pachymeter with a resolution of 0.05 mm. The mass of the ball was measured using a digital scale with a resolution of 0.001g.

Table 1. Geometric characteristics of the tennis balls

Brand	Condition	Diameter (10 ⁻³ m)	Mass (10 ⁻³ kg)	Seam width (10 ⁻³ m)
WILSON 3	New	66.1(0)	57.74(2)	1.8(5)
WILSON 3	Used	63.9(1)	55.09(7)	2.0(1)
BABOLAT TROPHY	New	64.7(3)	56.70(9)	3.6(8)
BABOLAT TROPHY	Used	63.2(0)	54.41(4)	3.2(6)

The drag coefficient, C_D , and the Reynolds number, Re , were set in accordance with Fox (2006):

$$C_D = \left[\frac{F_D}{\frac{1}{2} \rho V^2 A} \right] \quad (2)$$

$$Re = \frac{VL}{\nu} \quad (3)$$

In which F_D is the drag force on a tennis ball, ρ is the air density, 1.18 Kg/m³, V is the average air speed, A is the frontal area of the object, i.e., the tennis ball, L is the characteristic length. For a tennis ball, it is the diameter defined in Tab. 1, and ν is the kinematic viscosity of the air at room temperature, 1.54x10⁻⁵ m²/s.

3. RESULTS AND DISCUSSION

3.1. Position effect

Initially, the tennis balls were compared in relation to their position in the test section, i.e., the effect of the position of the seam (kerfs) on the aerodynamic properties was assessed. Each new and used ball was analyzed, in the test

section, attached in the positions 1 and 2, as shown in Fig. 2, 3, 4 and 5. Position 2 is obtained by rotating position 1 in 90° . For the other possible positions, it was considered the equivalent to one of these, in other words, 180° is equivalent to the rotated position of 0° . The 270° position is equivalent to the symmetrical of the position 2 (90°) in relation to a horizontal axis, passing through the center of the ball.

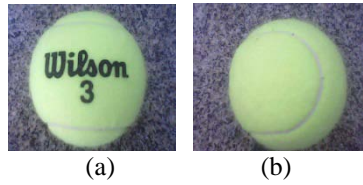


Figure 2. Wilson Tennis Ball 3(New), (a) position 1 e (b) position 2

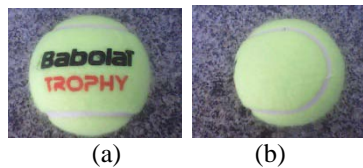


Figure 3. Babolat Tennis Ball (New), (a) position 1 and (b) position 2



Figure 4. Babolat Tennis Ball (Used), (a) position 1 and (b) position 2

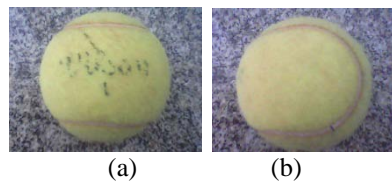


Figure 5. Wilson Tennis Ball 3 (Used), (a) position 1 and (b) position 2

It is highlighted, from Fig. 4 and 5, that it is impossible to measure the time of use of each ball analyzed in the test section of the wind tunnel.

Figures 6, 7, 8, and 9 represent the drag coefficient versus the Reynolds number to the new and used ball in the positions 1 and 2.

The drag coefficient and Reynolds number shown in these figures were calculated by the Eq. (3) and (2).

In these figures, the drag coefficient depends on the Reynolds number. Therefore, the effect of compressibility or the free surface effect on the drag force was neglected.

Aerodynamic studies establish relations between the drag coefficient and the Reynolds number through a dimensional analysis. The Reynolds number is a dimensionless number in which the spreading speed of the fluid on the object is related to the geometry of the object.

Fig. 6 and 7 show that the drag coefficient, regardless of the position in which the ball was placed in the test section, decreases with the increase of the Reynolds number, as observed in the theory. There is non-significant difference between positions 1 and 2. This difference, however, could not possibly affect the parameter position, for the new balls. For both balls, it is observed similarity in the curve of the drag coefficient versus the Reynolds number, i.e., there is no difference between the Wilson Ball 3 and the Babolat ball. This demonstrates the presence of frictional and pressure drags in both balls.

Figures 8 and 9 are qualitatively analogous to Fig. 6 and 7, as previously described. However, it is very clear that there is a difference in Fig. 9 related to the position. This figure shows higher wear and tear of this ball when compared to the ball analyzed in Fig. 8.

Figure 8 presents a drag coefficient inferior to 3.5 to $Re = 2 \times 10^4$, regardless the position. It is observed in Fig. 9.

The numerical value of the drag coefficient is higher in Fig. 6 and 7, compared to Fig. 8 and 9, due to the presence of fuzz.

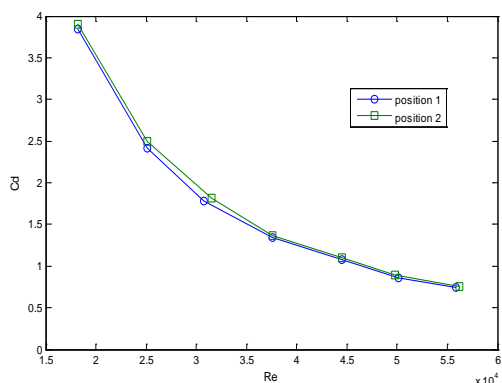


Figure 6. Re versus C_D to BABOLAT ball (New)

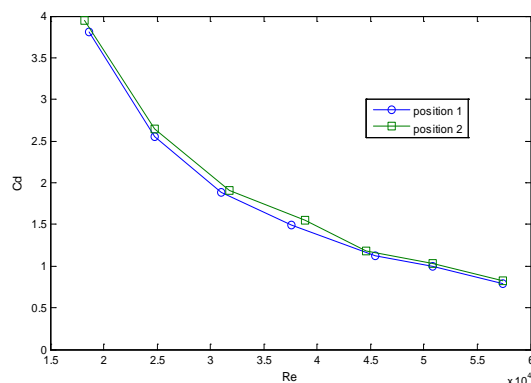


Figure 7. Re versus C_D to the WILSON Ball 3 (New)

It is noteworthy that fuzz tends to annul the effects of the position of the seam. Usually, the Position 2, whose frontal area presented a larger area length of sewing, showed higher drag values for lower Reynolds number in all examined balls.

Fig. 6, 7, 8, and 9 also highlight that, for high Reynolds numbers ($> 3.5 \times 10^4$), lower drag coefficient is found, which suggests the loss of the influence of the position of the seam.

This finding was observed in the studies conducted by Mehta and Pallis (2001a, 2001b) to the Re between 46000 and 161000 in two Wilson tennis balls.

The value of the drag coefficient obtained by Mehta and Pallis (2001a, 2001b) is significantly similar to those in Fig. 8 and 9, i.e., $Re = 46000$.

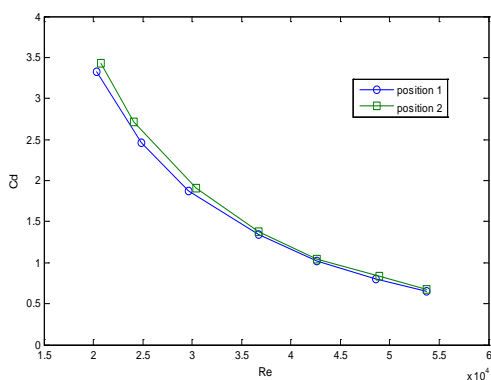


Figure 8. Re versus C_D to the BABOLAT ball (Used)

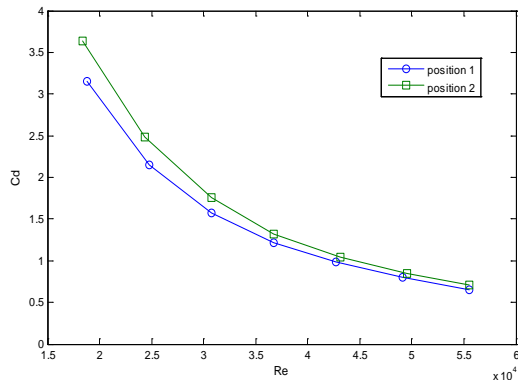


Figure 9. Re versus C_D WILSON 3 ball (Used)

Figures 8 and 9 show a maximum average variation of 9% of reduction in the drag from position 1 to position 2, to the fuzzed ball. However, an average of 8% of variance was found for the drag coefficient, due to the orientation of the seam for low Reynolds numbers (below 80km/h), in the studies of Alam *et al.* (2003, 2004b), which is an acceptable difference. These high percentages of reduction are due to the fact that the position number 1 gives a more aerodynamic shape to the ball, because of the presence of more favorable lines to the runoff, reducing the turbulent boundary layer and delaying displacement. In the balls used in this study, the difference of position was more evident, mainly in Fig. 9. This phenomenon can be attributed to the lack of fuzz, which makes the presence of the seam more noticeable.

The high values of C_D for low Re leads us to discard this trend and ascribe it to an experimental error, because they become difficult to be measured, since the flow of the wind tunnel is slowed down, which requires higher sensitivity from the acquisition system. However, Mehta *et al.* (2008) compared their results to that of a smooth sphere. The total error to the drag force of the tennis balls would be lower since the drag force is higher. Thus, they carried out an analysis on a smaller scale and initially attributed these high values to the change in orientation of the filaments, which, at slow or almost null speeds, remained almost perpendicular to the surface of the ball.

They consider that, as the flow velocity increases, the filaments are forced to bend down due to aerodynamic effects of drag. Therefore, the contribution of drag is reduced due to a high Reynolds number. Besides the inclination in favor of the runoff, Mehta *et al.* (2008) re-examines each filament in relation to the Reynolds number. It is estimated that, based on the filament diameter, the estimated Reynolds number is around 20, which places it at a range in which the C_D (considering the circular wire-cylinder) is high ($C_D \approx 3$) and inversely proportional to the Reynolds number. Therefore, high values of the drag coefficient for low Reynolds numbers are assigned to the combined effect of the fuzz filament orientation and the Reynolds numbers associated with individual filaments. Therefore, it explains the achievement of the drag coefficient, C_D , ranging up to 4 for low Re , as shown in Fig. 6, 7, 8, and 9.

It is noteworthy that several investigators achieved no results for speeds below 50 km/h, as evidenced in Fig. 6, 7, 8 and 9.

Fig. 6 and 7 show a drag coefficient of around 0.60 for high values of Re . This fact was observed by several researchers, including Alam *et al.* (2004a, 2004c), who determined values of drag coefficient ranging from 0.55 to 0.65 for most new tennis balls, and Mehta and Pallis (2001), who found a drag coefficient of around 0.62 for new balls.

3.2. Effect of fuzz

It is not possible to carry out a direct comparison between the used and new balls for the same position with regard to fuzz, since the diameters of the balls are different and the diameter is compared by a separated study. This difference in diameter is a result of the use of these balls in competition, i.e., the balls reduce their diameter throughout their useful life due to internal pressure loss. Therefore, the comparison between balls was performed considering the same brand, position and diameter and the results are defined for balls that only differ in the presence or absence of fuzz. The methodology used in this analysis consists of rubbing the new balls, i.e., removing the fuzz, performing tests on the tunnel and comparing the results with those already obtained for the balls before the rubbing process. Figure 10 illustrates the Reynolds number versus the drag coefficient, for the analysis of the fuzz effect.

The fuzzless Wilson 3 balls analyzed in the experiment presented a slight decrease in the drag coefficient, as shown in Fig. 10. The comparison between fuzzless Wilson 3 balls and the same type of balls with fuzz, reduced drag coefficient to a value close to 10.2% is observed for low values of Reynolds number.

In the manufacturing process, on the cover tissue of a tennis ball, the junctions of the elements of fuzz on its surface define the relative roughness on the surface of the felt, which is evident by simple observation. The elements of fuzz have finite thickness and length, thus forming, as defined Mehta *et al.* (2008), an additional porous coat in the ball, through which the air can flow.

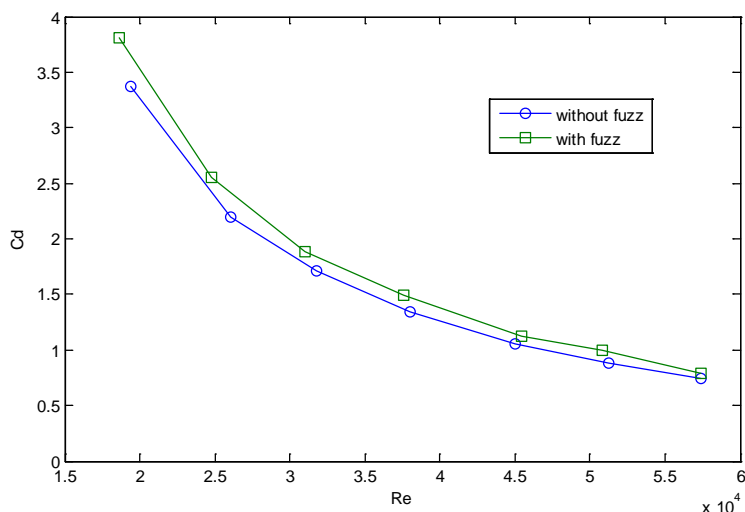


Figure 10. Re versus C_D for new WILSON 3 with and without fuzz

Thus, a tennis ball can be seen as a rough sphere with a porous cover. Subsequently, each element of fuzz will also experience a drag pressure. Therefore, Mehta *et al.* (2008) define "fuzz drag" as the sum of drag pressure experienced by each element of fuzz on the surface of the ball.

3.3. Effect of diameter

A new fuzzless Wilson 3 ball and a used Wilson 3 ball (fuzz removed by usage) were used in this analysis, as shown in Fig. 11. According to Tab. 1, the analyzed balls present a difference of approximately 3.3% in diameter. With this difference, it is possible to estimate an increase of almost 7% in the value of the drag coefficient, according to Eq. (2). Uncorrected experimental values indicated an approximate increase of 9% in the diameter. By correction calculation, Eq. (1), with an area of test section of 32.300 mm², using the diameters from Tab. 1 and Eq. (1), the correction coefficients 0.952 and 0.949 were achieved, respectively, to the used and the new ball. Thus, the average percentage change in diameter is now almost 8% (7.86%). Evidently, if the diameter of the ball is larger, the drag force will be increased due to the higher projection of the frontal area, but a simple range of sizes should not affect C_D , since the other parameters, such as surface characteristics and fuzz have not changed significantly.

With roughness, the growth rate of the boundary layer is increased, resulting in early detachment and consequently higher C_D values. For high values of Re , according to Mehta *et al.* (2008), C_D is expected to reach a constant level and increase with increased roughness, in the transcritical flow, as evidenced in the measurements

performed by Achenbach (1974). However, the same data from Achenbach (1974) demonstrate an upper limit for $C_D \approx 0.4$ for spheres with increasing roughness (transcritical flow). According to Achenbach, the measure for the location of separation for this C_D value is approximately 100° . Increased superficial friction coefficient makes the boundary layer more susceptible to separation, in opposition to the tendency of the boundary layer of separating as it becomes thicker (Mehta *et al.*, 2008).

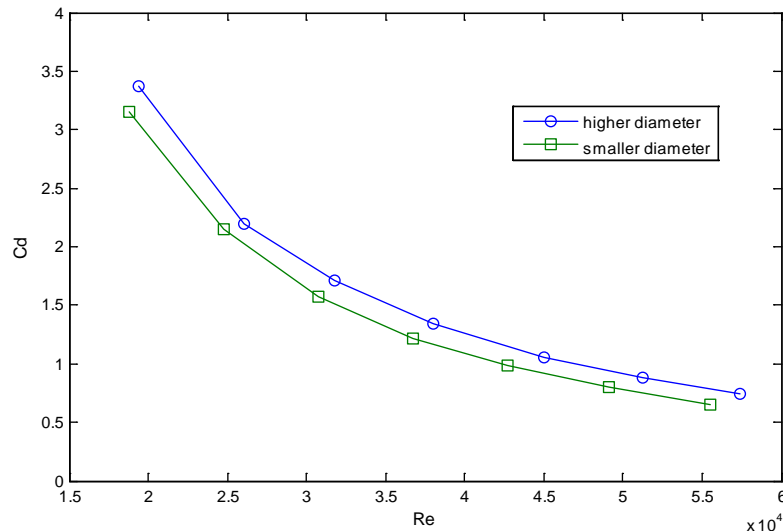


Figure 11. Re versus new and used C_D WILSON (without fuzz).

Although transcritical flow is not in the scope of this research, it is noteworthy that, for certain types of roughness, a limit is reached for a C_D in this condition because the effects of increased thickness of the boundary layer are subjugated to those due to the increased superficial friction coefficient.

4. CONCLUSION

The present work was based on low-speed aerodynamics, in which roughness effects can be represented. The conclusions achieved by this work support evidence that surface roughness factor, represented in this work by hair and seams, is significant under conditions of low Re values. The results presented here indicate that roughness is a factor for increased drag, while saliencies in golf balls induce turbulence in the limit layer, thus delaying detachment and reducing drag.

In tennis games, the ball moves with a speed ranging of between $40000 < Re < 400000$ of Reynolds numbers. However, the main advantage observed of operating at levels below this interval or at its beginning is the observation of fuzz behavior in the early stages of the runoff, as stated in item *Discussion*. Thus, the flow over a tennis ball usually happens in the transcritical flow, in which the location of detachment moves significantly depending on the Reynolds number. According to Mehta *et al.* (2008), it means that C_D does not depend on the Reynolds number, since the total drag on a rounded body, such as a tennis ball, is almost completely caused by the pressure drag.

The high values of the drag coefficient is attributed to low Reynolds numbers, a combined effect of the orientation of the ball fuzz filaments and the effects of Reynolds numbers for each individual filament.

In the present study, the drag coefficient of the analyzed balls ranged between $C_D \approx 3$ and $C_D \approx 0.50$, for low Reynolds numbers. The average drag coefficient varies between 0.55 and 0.65 for a new tennis ball for Reynolds numbers ranging between $69000 < Re < 161000$ (60-140 km/h). However, the C_D value for a used ball is slightly lower compared to that of a new ball, for the reasons already stated. The orientation of the seam has a negligible effect on the drag coefficient at high Reynolds numbers. However, some effects were observed for low Reynolds numbers ($\sim 9\%$ increase in the C_D value).

The fuzz on a tennis ball causes an early transition on the laminar boundary layer and a rapid increase in the thickness of the turbulent boundary layer. This implies that the dislocation moves towards the apical region (top of ball), comparing it with that of the laminar boundary layer for subcritical Reynolds numbers, as explained by Mehta *et al.* (2008).

Although the drag coefficient in the bigger ball is comparable to smaller balls, it is obviously higher due to larger cross-section area. The results revealed that it is possible to achieve the effect desired by ITF and other associations related to tennis, namely, the slowing of the pace of a game (by increasing the time of the movement of the ball).

Even operating with the wind tunnel with high blocking rates, the results were conclusive and relevant after the performance of the correction.

In this research, the influence of seam on the aerodynamic properties of tennis balls was not clear for Reynolds numbers lower than 60000. Many researchers, including Mehta *et al.* (2008), demonstrated that, for Reynolds values higher than 150000, the data in the transcritical flow, for each ball, can be calculated in such a way to provide a single value of the drag coefficient. Studies by Alam *et al.* (2004) follow this idea for high values of Reynolds: the results revealed that the orientation of the seam has little effect for high Re (92000 or higher than 80km/h).

The total drag of bodies with geometry similar to that of a tennis ball is composed of pressure drag (due the different pressure values between the front and rear) and a small part by viscous drag (non-slip condition). Achenbach (1972) estimated that the viscous drag for a smooth sphere, close to the transcritical flow (fully turbulent boundary layer), is approximately 2% of the total drag. Therefore, a major part of the total drag results from the pressure drag, a factor directly linked to the local detachment of the boundary layer. According to Mehta *et al.* (2008), high values of drag coefficient (higher than 0.5) are generated in the transcritical flow (or in this case for low Re). Following the transition, the place of displacement and transition slowly begins to rise, while the drag coefficient increases. At a certain point, the location of transition moves to the location of stagnation and the location of separation is then fully determined by the development of the turbulent boundary layer. For values of Reynolds above the operating range of this research, Mehta *et al.* (2008) attribute this difference in the drag coefficient to the techniques used to measure the diameter of the ball. The high values of the drag coefficient are, therefore, explained by the prevalence of the pressure drag and the fact that the location of turbulent boundary layer separation for tennis balls seems to be compared to those seen for laminar separation at relatively low Re , which explains the values of drag coefficient approximately equal to 3 for low Reynolds numbers.

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