# STIFFNESS ANALYSIS OF A 6-RSS PARALLEL MANIPULATOR

**Rogério Sales Gonçalves, rsgoncalves@mecanica.ufu.br João Carlos Mendes Carvalho, jcmendes@mecanica.ufu.br** Faculdade de Engenharia Mecânica, Universidade Federal de Uberlândia, Campus Santa Mônica, CEP 38400-902, Uberlândia – MG – Brasil

Abstract. This paper describes the stiffness analysis of a  $6-\underline{RSS}$  parallel manipulator. The  $6-\underline{RSS}$  kinematic chain is composed by six legs where the axis of the actuators are placed two-by-two along the three axis of a Cartesian reference frame. The manipulation performances of a robot are directly affected by the stiffness structure because external forces and moments may cause large deflections in the links and joints, which are undesirable from the viewpoint of both accuracy and payload performances. The stiffness analysis is based on standard concepts of static elastic deformations. Its analysis can be carried through from the stiffness matrix that can be numerically computed by defining a suitable model of the manipulator, which takes into account the stiffness properties of each element such as links, actuators and joints. In this paper the Jacobian matrix is obtained from the  $6-\underline{RSS}$  structure kinematic model and then, its stiffness matrix too. A general scheme of the strucuture considering all elements stiffness (links and joints) is presented. In order to simplify the model, the joints stiffness have been neglected. The obtained stiffness matrix is used to map the end-effector displacement when external wrench are applied on it. The used methodology can be applied to analyze the stiffness of other robots structures.

Keywords: Parallel Manipulators, stiffness analysis, compliance matrix.

## **1. INTRODUCTION**

A load applied on a body produces changes on its geometry that are known as deformations or compliances displacements. Stiffness can be defined as the capacity of a mechanical system to sustain loads without excessive changes of its geometry (Rivin, 1999). Moreover, the stiffness of a body can be defined as the amount of force that can be applied per unit of compliance displacement of the body (Nof, 1985), or the ratio of a steady force acting on a deformable elastic medium to the resulting displacement (AccessScience, 2003).

Compliance displacements in a parallel robotic system improve for mechanical float of the end-effector relative to the fixed base. This produces negative effects on static and fadigue strength, wear resistance, efficiency (friction losses), accuracy, and dynamic stability (vibration). The increasing importance of high accuracy and dynamic performance for parallel robotic systems has increased the use of high strength materials and lightweight designs by giving significant reduction of cross-sections and weight. Nevertheless, these solutions also increase structural deformations and may result in intense resonance and self-excited vibrations at high speed (Rivin, 1999). Therefore, a study of stiffness becomes of primary importance in the design of multibody robotic systems in order to properly choose materials, component geometry, shape and size, and interaction of each component with others. Some examples of design procedures based on stiffness analysis can be found in (Liu *et al.*, 2000) (Simaan and Shoham, 2002) (Carbone *et al.*, 2003). In particular, Liu et al. propose an optimum design procedure for 3 degree of freedom spherical parallel manipulators as based on the computation of conditioning and stiffness indices. Simaan and Shoham (2002) propose an optimum design procedure for the synthesis of a planar robot having variable geometry as based on stiffness performance. Carbone et al. (2003) propose a procedure for optimal design of a biped leg module for humanoid robots by considering stiffness performance and lightweight design.

A stiffness analysis can be also very useful for estimating the expected performance of a system in term of payload and accuracy and for verifying the feasibility of specific tasks as proposed for example in (Pai and Leu, 1991) (Carbone *et al.*, 2002) and (Ceccarelli *et al.*, 2002).

The overall stiffness of a manipulator depends on several factors, including the size and material used for the links, the mechanical transmission mechanisms, the actuators, and the controller (Tsai, 1999). In general, to realize a high stiffness mechanism, many parts should be large and heavy. However, to achieve high speed motion, these should be small and light. Moreover, one should point out that the stiffness is greatly affected by both the position and the values of the mechanical parameters of the structures parts (Yoon *et al.*, 2004).

For the stiffness analysis is necessary obtaining the stiffness model of the parallel structure. In the literature there are three main methods have been used to derive the stiffness model of parallel manipulators (Deblaise *et al.*, 2006). These methods are respectively based on: the calculation of the Jacobian matrix (Gosselin, 1990) (Khasawneh and Ferreira, 1999) (Zhang *et al.*, 2004) (Zhang, 2004) (Gosselin and Zhang, 2002) (Majou *et al.*, 2004) (Company *et al.*, 2005); the Finite Element Analysis (Corradini *et al.*, 2004) (Bouzgarrou *et al.*, 2004) ) and the matrix structural analysis (Huang *et al.*, 2002) (Li *et al.*, 2002) (Martin, 1966) (Wang, 1966) (Imbert, 1979) (Clinton *et al.*, 1997) (Dong *et al.*, 2005)

The methods based on calculation of the Jacobian matrix are simple and they supply one initial estimate of the stiffness matrix. One of the first stiffness analysis has been conducted by Gosselin (1990) and the stiffness of a Parallel Kinematic Machines is mapped onto its actuated joints whereas links are supposed as perfectly rigid. This approach was also involved by Khasawneh and Ferreira (1999) and Zhang et al. (2004). To take into account the link flexibility, this first model was supplemented by the lumped model developed by Zhang (2004). Flexible links are then replaced by rigid beams mounted on revolute joints plus torsional spring located at the existing joints.

The uses of Finite Element Analysis models are reliable, but these models have to be remeshed over again, these results in very tedious and time-consuming routines (Huang *et al.*, 2002). However these models are well adapted to validate analytical models (Khasawneh and Ferreira, 1999) (Li *et al.*, 2002), or some experimental results (Corradine *et al.*, 2004). Bouzgarrou *et al.* (2004) used the Finite Element Analysis model to study static rigidity and natural frequencies of the T3R1 parallel robot.

Another methods based on matrix structural analysis are not so often used. Clinton *et al.* (1997) used this approach to derive the stiffness matrix for each element of a Stewart platform and then assemble individual ones into a system stiffness matrix. This approach is also used in Huang *et al.* (2002) to give the stiffness model of a machine frame considered as a substructure. The superposition principle is used to achieve the stiffness model of the machine structure as a whole.

In this paper is presented the calculated stiffness matrix of the 6- $\underline{R}SS$  parallel manipulator. This matrix can be computed based on lumped stiffness parameters of the links. The Jacobian matrix is obtained from the kinematic model of the 6- $\underline{R}SS$  parallel manipulator and then the stiffness matrix is calculated. The end-effector output forces are related to its deflections by the stiffness matrix. With the stiffness matrix one do the map of the compliance displacement of end-effector as function of the external efforts applied on its center, and in this paper it is carried through for different configurations of 6- $\underline{R}SS$  parallel manipulator.

### 2. THE 6-RSS PARALLEL ARCHITECTURE

The 6-<u>R</u>SS parallel architecture is a 6 degree of freedom manipulator, which is characterized by a base and a platform (end-effector), connected by six *RS-SS* segments, where the R-joint is on the Cartesian axes, two joints by axis, as shown in Fig. 1. The Cartesian system is the base of the structure where the actuators are mounted, reducing dead loads. The links can be constructed by light and resistant materials that make then rigid, with low inertia and in a modular construction. The *S*-joints on the other extremity of the segments are connected at the end-effector, consisting in a virtual cube where the *S*-joints are tied on the center of its faces, having then, three crossed segments. Kinematics variables are the input angles  $\alpha_i$  (*i*=1 to 6) in the *R*-joints. The studied structure has the *RS*-segment and the *SS*-segment the same length and:  $|b_1b_2| = |b_3b_4| = |b_5b_6| = |p_1p_2| = |p_3p_4| = |p_5p_6|$ , Fig. 1.



Figure 1. (a) Initial configuration of the 6-<u>R</u>SS parallel architecture; (b) Generic configuration.

Figures 2 and 3 show the prototype constructed at the Laboratory of Automation and Robotics in Uberlândia/Brazil.



Figure 2. The built prototype of the 6-RSS Cartesian parallel manipulator.



(a) Detail of legs and actuators.



(b) End-effector with a linear transducer.

Figure 3. Detail of the prototype of a 6-<u>R</u>SS Parallel Manipulator.

# 2.1. Inverse Kinematic Model

Kinematic equations can be obtained by using a suitable analysis procedure with a vector and matrix formulation (Bezerra, 1996). Equations can be written by considering a generic reference frame fixed to the base ( $O_F$ ,  $x_F$ ,  $y_F$ ,  $z_F$ ) and another at the platform ( $O_M$ ,  $x_M$ ,  $y_M$ ,  $z_M$ ) as shown in Fig. 4. The parameters are:  $\alpha_i$  are the generalized coordinates;  $b_i$  the position vector of the base points  $b_i$  (position of the R-joint in the base),  $S_i$  and  $L_i$  the arm and the forearm vectors, all referred to the base reference frame, and  $S_i$  and  $L_i$  their lengths;  $p_i$  the position vector of the platform points  $p_i$  (position of the S-joint at the mobile platform), referred to the mobile reference frame. All parameters for i=1 to 6.



Figure 4. Vectors for modeling the mechanism.

From Figure 4 one can obtain:

$$\mathbf{L}_{i} = \begin{bmatrix} R \end{bmatrix} \mathbf{p}_{i} + \mathbf{t} - \mathbf{b}_{i} - \mathbf{S}_{i} \qquad \text{for } \mathbf{i} = 1 \text{ to } \mathbf{6} \tag{1}$$

where

$$b_{i} = \begin{cases} b_{ix} & b_{iy} & b_{iz} \end{cases}^{b} \\ p_{i} = \{ p_{ix} & p_{iy} & p_{iz} \}^{b} \\ S_{i} = \{ S_{ix} & S_{iy} & S_{iz} \}^{b} \\ L_{i} = \{ l_{ix} & l_{iy} & l_{iz} \}^{b} \\ S_{i} = |S_{i}| = (S_{ix}^{2} + S_{iy}^{2} + S_{iz}^{2})^{1/2} \\ L_{i} = |L_{i}| = (l_{ix}^{2} + l_{iy}^{2} + l_{iz}^{2})^{1/2} \\ R_{i} = |I_{i}| = (l_{ix}^{2} + l_{iy}^{2} + l_{iz}^{2})^{1/2} \\ [R] = [n \quad o \quad a] = \begin{bmatrix} n_{x} & o_{x} & a_{x} \\ n_{y} & o_{y} & a_{y} \\ n_{z} & o_{z} & a_{z} \end{bmatrix}$$
for i=1 to 6 (2)

In Equation (2) only the length  $L_i$  of the vector  $L_i$  is known. Then, one can write:

$$L_{i}^{2} = ([R]p_{i} + t - b_{i} - S_{i})^{2}$$
 for i=1 to 6 (3)

Developing Eq. (3), one have:

$$2p_{ix}[(n_{x}t_{x} + n_{y}t_{y} + n_{z}t_{z} - n_{x}S_{ix} - n_{y}S_{iy} - n_{z}S_{iz})] + 2p_{iy}[(o_{x}t_{x} + o_{y}t_{y} + o_{z}t_{z} - o_{x}S_{ix} - o_{y}S_{iy} - o_{z}S_{iz})] + 2p_{iz}[(a_{x}t_{x} + a_{y}t_{y} + a_{z}t_{z} - a_{x}S_{ix} - a_{y}S_{iy} - a_{z}S_{iz})] + 2b_{ix}[(S_{ix} - t_{x} - n_{x}p_{ix} - o_{x}p_{iy} - a_{x}p_{iz})] + 2b_{iy}[(S_{iy} - t_{y} - n_{y}p_{ix} - o_{y}p_{iy} - a_{y}p_{iz})] + 2b_{iz}[(S_{iz} - t_{z} - n_{z}p_{ix} - o_{z}p_{iy} - a_{z}p_{iz})] + (t_{x}^{2} + t_{y}^{2} + t_{z}^{2}) + p_{i}^{2} + S_{i}^{2} + b_{i}^{2} - 2(t_{x}S_{ix} + t_{y}S_{iy} + t_{z}S_{iz}) - L_{i}^{2} = 0$$

$$(4)$$

Choosing proper reference frames, Eq. (4) can be simplified. In this work the used reference frames are:  $(X_b, Y_b, Z_b)$  at the base, where the origin coincides with point *O*;  $(X_p, Y_p, Z_p)$  at the center of mobile platform as represented in Fig. 1. The matrix of rotation [*R*] is given by the Euler angles (rotation  $\phi_1$  about *x*, rotation  $\phi_2$  about *y* and rotation  $\phi_3$  about *z*):

$$\begin{bmatrix} R_{Euler} \end{bmatrix} = \begin{bmatrix} c\phi_2 c\phi_3 & -c\phi_2 s\phi_3 & s\phi_2 \\ s\phi_1 s\phi_2 c\phi_3 + c\phi_1 s\phi_3 & -s\phi_1 s\phi_2 s\phi_3 + c\phi_1 c\phi_3 & -s\phi_1 c\phi_2 \\ -c\phi_1 s\phi_2 c\phi_3 + s\phi_1 s\phi_3 & c\phi_1 s\phi_2 s\phi_3 + s\phi_1 c\phi_3 & c\phi_1 c\phi_2 \end{bmatrix}$$
(5)

The inverse kinematic model of the 6-<u>RSS</u> parallel manipulator obtains the co-ordinates  $\alpha_i$  (i=1 to 6) when the position  $(t_x, t_y, t_z)$  and orientation  $(\phi_l, \phi_2, \phi_3)$  of the platform are known. Equation (4) constitute a system with six variables:  $\alpha_1$ ,  $\alpha_2$ ,  $\alpha_3$ ,  $\alpha_4$ ,  $\alpha_5$  and  $\alpha_6$ , representing the inverse kinematic model of the 6-<u>RSS</u> parallel manipulator. In this system, vectors  $p_i$ ,  $b_i$  and t, the matrix [R] and lengths of  $S_i$  and  $L_i$  vectors are known. Only the generalized coordinates  $\alpha_i$  (i=1 to 6) are unknown.

This system has 64 solutions in general. But the system solution has repeated roots, including configurations not possible to be attained by the mechanism, having then only one set of roots  $\alpha_i$  (i=1 to 6) valid (Bezerra, 1996) (Carvalho, et al., 2001).

# 3. STIFFNESS ANALYSIS OF 6-RSS PARALLEL MANIPULATOR

#### 3.1. Stiffness Model

The stiffness properties can be defined through a 6 x 6 matrix that is called as stiffness matrix *K*. This matrix gives the relationship between the vector of the compliance displacements  $\Delta S = \{\Delta x, \Delta y, \Delta z, \Delta \phi_1, \Delta \phi_2, \Delta \phi_3\}^T$  occurring to a frame fixed at the end of the kinematic chain when a static wrench  $W = \{Fx, Fy, Fz, T_x, T_y, T_z\}^T$  acts upon it (Tsai, 1999). The relationship between the vector of the compliant displacement  $\Delta S$  and external wrench *W* can be written in the form, Eq. (6):

$$W = K \Delta S \tag{6}$$

The stiffness matrix can be numerically computed by defining a suitable model of the robot, which takes into account the stiffness properties of each element of the robot and, in particular, the lumped stiffness parameters are modeled as linear and torsion springs. A simplified stiffness model of  $6-\underline{R}SS$  Parallel Manipulator can be defined as show in Fig. 5. It is worthy to note that the stiffness matrix is configuration dependent since it is a function of the input angles. Therefore, the solution of the inverse kinematics is necessary.



Figure 5. Stiffness Model for the 6-RSS Parallel Manipulator.

In Figure 5 the spherical joints are changed for torsion spring, where  $T_i$  (i = 1 to 6) are the torsion springs in place of revolute joints;  $k_{fi}$  and  $k_{ai}$  (i = 1 to 6) are linear spring that represent the forearm and arm, respectively. Finally  $T_{fi}$  and  $T_{ai}$  (i = 1 to 6) represent the torsion spring of the spherical joints at the extremity of segments RS and SS respectively.

#### 3.2. Jacobian Matrix

Several different approaches can be used for deriving the stiffness matrix. A first approach is based on the computation of the Jacobian matrix for the given multibody robotic system. With the inverse kinematic model of the architecture it can write:

$$\boldsymbol{\alpha} = \mathbf{G} \, \mathbf{S} \tag{7}$$

where  $\alpha$  is the vector of the joint coordinates and  $\mathbf{S} = [t_x t_y t_z \phi_1 \phi_2 \phi_3]^T$  is the vector expressing position and orientation of a reference frame attached to the end-effector with respect to the fixed reference frame; **G** is a matrix expressing the inverse kinematics. By using Taylor series expansion for Eq. (7) and considering only its first term, one can write:

$$\Delta \alpha = \mathbf{J}_{\mathbf{x}} \, \Delta \mathbf{S} \tag{8}$$

where  $\Delta \alpha$  is the vector of increments of the joint coordinates,  $\Delta S$  is the vector of increment of the  $t_x, t_y, t_z, \phi_1, \phi_2$  and  $\phi_3$  and  $J_x$  is the Jacobian matrix relative of the inverse kinematic model. The Jacobian matrix can be written in form:

$$\mathbf{J}_{\mathbf{x}} = [\mathbf{J}_1, \mathbf{J}_2, \dots, \mathbf{J}_6] \tag{9}$$

where

$$\mathbf{J}_{1} = \left(\frac{\partial \alpha_{1}}{\partial t_{x}}, ..., \frac{\partial \alpha_{6}}{\partial t_{x}}\right)^{T} \quad \mathbf{J}_{2} = \left(\frac{\partial \alpha_{1}}{\partial t_{y}}, ..., \frac{\partial \alpha_{6}}{\partial t_{y}}\right)^{T} \quad \mathbf{J}_{3} = \left(\frac{\partial \alpha_{1}}{\partial t_{z}}, ..., \frac{\partial \alpha_{6}}{\partial t_{z}}\right)^{T}$$

$$\mathbf{J}_{4} = \left(\frac{\partial \alpha_{1}}{\partial \phi_{1}}, ..., \frac{\partial \alpha_{6}}{\partial \phi_{1}}\right)^{T} \quad \mathbf{J}_{5} = \left(\frac{\partial \alpha_{1}}{\partial \phi_{2}}, ..., \frac{\partial \alpha_{6}}{\partial \phi_{2}}\right)^{T} \quad \mathbf{J}_{6} = \left(\frac{\partial \alpha_{1}}{\partial \phi_{3}}, ..., \frac{\partial \alpha_{6}}{\partial \phi_{3}}\right)^{T}$$

$$(10)$$

By applying the principle of duality between the force/torque and velocity fields (Bashar et al., 1999), one can write:

$$W = J_x^T F \tag{11}$$

where  $F = \{F_{xi}, F_{yi}, F_{zi}, T_{xi}, T_{yi}, T_{zi}\}^T$  is the vector of forces acting on each link;  $J_x^T$  is the transpose matrix of the Jacobian defined in Eqs. (9) and (10). Moreover, neglecting the effects in the joints and considering Fig. 5

$$\mathbf{F} = \mathbf{K}_{\mathrm{L}} \,\Delta \alpha \tag{12}$$

where

$$K_{L} = \begin{bmatrix} k_{1} & 0 & 0 & 0 & 0 & 0 \\ 0 & k_{2} & 0 & 0 & 0 & 0 \\ 0 & 0 & k_{3} & 0 & 0 & 0 \\ 0 & 0 & 0 & k_{4} & 0 & 0 \\ 0 & 0 & 0 & 0 & k_{5} & 0 \\ 0 & 0 & 0 & 0 & 0 & k_{6} \end{bmatrix}$$
(13)

where  $k_i$  (i = 1,...,6) are the lumped stiffness parameters of the *i*-th link. From Equations (8) and (12)

$$\mathbf{F} = \mathbf{K}_{\mathrm{L}} \, \mathbf{J}_{\mathrm{x}} \, \Delta \mathbf{S} \tag{14}$$

Multiplying both sides of Eq. (14) by  $J_x^T$  and substituting in Eq. (11) one have

$$W = J_x^T \mathbf{K}_{\mathrm{L}} \, \mathbf{J}_{\mathrm{x}} \, \Delta S \tag{15}$$

In this way, from Eqs. (6) and (15) the stiffness matrix of 6-RSS parallel manipulator can be defined as:

$$K = J_x^T \mathbf{K}_{\mathbf{L}} \mathbf{J}_{\mathbf{x}}$$
(16)

#### 4. NUMERICAL ESTIMATION OF STIFFNESS PERFORMANCES

From Figure 5, the 6-<u>R</u>SS parallel manipulator has six set of equals forearm and arm. Therefore, Eq. (16) can be simplified considering a same lumped stiffness parameters k ( $K_L = diag(k)$ )

$$K = k J_x^T J_x$$
<sup>(17)</sup>

Figure 6 show one set of forearm and arm of the  $6-\underline{R}SS$  parallel manipulator with the lumped stiffness parameters, considering only the stiffness of the forearm and arm:



Figure 6. Lumped Stiffness Parameters.

For the structure  $6-\underline{R}SS$  the forearm and arm have the same length. Therefore, referring to Fig. 6 the equivalent lumped stiffness parameter for each leg can be write:

$$k_i^{-1} = k_{fi}^{-1} + k_{ai}^{-1} \quad \text{for } (i = 1, ..., 6)$$
(18)

The lumped stiffness parameters  $k_{fi}$  and  $k_{ai}$  are calculated as:

$$k_{fi} = k_{ai} = \frac{E_i A_i}{L_i} \tag{19}$$

where  $E_i$  is the Young's module of the material,  $A_i$  is the section of the link and  $L_i$  is its length. Based on prototype the scalar lumped stiffness parameter k is 1,056e7 N/m. Using Equation (16) and (6) the compliance displacement for a given external wrench can be computed as:

$$\Delta \mathbf{S} = \mathbf{K}^{-1} \mathbf{W} \tag{20}$$

By using the proposed formulation the compliances displacements, which are due to the external force and torque can be numerically computed. Tables 1, 2 and 3 show the computed values of the displacements of  $6-\underline{R}SS$  parallel manipulator in three different configurations. The Figure 7 plots the compliance displacement in function of an applied external wrench. This proposed formulation can be also used to compute the extreme values of the stiffness over the workspace of the manipulator.

$\Delta x_{6-RSS}$	Fe = (100, 0, 0)	Fe = (0, 100, 0)	Fe = (0, 0, 100)	Fe = (100,100, 100)
_	Te = (0, 0, 0)			
	[N; Nm]	[N; Nm]	[N; Nm]	[N; Nm]
∆tx [mm]	0.0526	0	0	0.0526
∆ty [mm]	0	0.0526	0	0.0526
∆tz [mm]	0	0	0.0526	0.0526
Δφ1 [°]	0	0	0	0
Δφ2 [°]	0	0	0	0
Δφ3 [°]	0	0	0	0

Table 1. Displacement of 6-<u>RSS</u> parallel manipulator when  $\alpha_1 = \alpha_2 = \alpha_3 = \alpha_4 = \alpha_5 = \alpha_6 = 0$  degree.

Table 2. Displacement of 6-<u>RSS</u> parallel manipulator when  $\alpha_1 = \alpha_2 = \alpha_3 = \alpha_4 = \alpha_5 = \alpha_6 = 10$  degree.

$\Delta x_{6-RSS}$	Fe = (100, 0, 0)	Fe = (0, 100, 0)	Fe = (0, 0, 100)	Fe = (100, 100, 100)
_	Te = (0, 0, 0)			
	[N; Nm]	[N; Nm]	[N; Nm]	[N; Nm]
Δtx [mm]	0.0629	-0.0156	-0.0156	0.0317
Δty [mm]	-0.0156	0.0629	-0.0156	0.0317
Δtz [mm]	-0.0156	-0.0156	0.0629	0.0317
∆¢1 [°]	0	0	0	0
Δφ2 [°]	0	0	0	0
Δφ3 [°]	0	0	0	0

Table 3. Displacement of 6-<u>RSS</u> parallel manipulator when  $\alpha_1 = 5$ ;  $\alpha_2 = -10$ ;  $\alpha_3 = 3$ ;  $\alpha_4 = 4$ ;  $\alpha_5 = 1$ ;  $\alpha_6 = 2$  degree.

	I		I	
$\Delta x_{6-RSS}$	Fe = (100, 0, 0)	Fe = (0, 100, 0)	Fe = (0, 0, 100)	Fe = (100, 100, 100)
_	Te = (1, 0, 0)	Te = (1, 0, 0)	Te = (1, 0, 0)	Te = (1, 1, 1)
	[N; Nm]	[N; Nm]	[N; Nm]	[N; Nm]
∆tx [mm]	-0.0970	-0.1193	-0.1191	0.1566
∆ty [mm]	-0.5817	-0.4009	-0.4680	0.4237
∆tz [mm]	-0.5300	-0.4165	-0.3780	0.4006
Δφ1 [°]	12.8729	10.2236	10.5190	-7.5639
Δφ2 [°]	6.2878	4.9498	5.1038	-4.1049
Δφ3 [°]	-18.9019	-14.9422	-15.3615	12.2677

From Table 1 for the initial condition ( $\alpha_1 = \alpha_2 = \alpha_3 = \alpha_4 = \alpha_5 = \alpha_6 = 0$ ) the applied Forces produce the same displacement in the direction of the force when other forces are zeros. If the same values of forces are applied the compliance displacement and angular displacement are equals. From Figure 7 if the external forces increase the displacements increase too.



Figure 7. Displacements of 6-<u>R</u>SS parallel manipulator as a function of Fx = Fy = Fz when Tx = Ty = Tz = 0 for  $\alpha_1 = 5$ ;  $\alpha_2 = -10$ ;  $\alpha_3 = 3$ ;  $\alpha_4 = 4$ ;  $\alpha_5 = 1$ ;  $\alpha_6 = 2$  degree. (a)  $\Delta tx$ ; (b)  $\Delta ty$ ; (c)  $\Delta tz$ ; (d)  $\Delta \phi 1$  [°]; (e)  $\Delta \phi 2$  [°]; (f)  $\Delta \phi 3$  [°]

## 5. CONCLUSION

In this paper the stiffness of  $6-\underline{R}SS$  parallel manipulator has been carried out as based on suitable kinematic and stiffness models. The Jacobian matrix is derivate from the inverse kinematic model of the structure. Then the stiffness matrix is calculated considering only the flexibility of the forearms and arms. The stiffness matrix enable to obtain the mobile platform displacement when an external wrench is applied. The procedure was used to map the mobile platform displacement for different configurations of the  $6-\underline{R}SS$  parallel structure. The used methodology can be applied to analyze the stiffness of other robots structures. Future works concern now in improving the stiffness model considering the joints and experimental test with the prototype.

#### 6. ACKNOWLEDGMENTS

The authors are thankful to CAPES for the financial support through which the research work above reported was partially supported.

#### 7. REFERENCES

AccessScience, 2003, "McGraw-Hill Dictionary of Scientific and Technical Terms", McGraw-Hill, available on line at <a href="http://www.accessscience.com/Dictionary/">http://www.accessscience.com/Dictionary/</a>>.

- Bezerra, C.A.D., 1996. Geometric Model of a Fully Cartesian Parallel Structure (In Portuguese). MSc Thesis, Federal University of Uberlândia, Brazil.
- Bouzgarrou, B. C., Fauroux, J. C., Gogu, G. and Heerah, Y., 2004, "Rigidity analysis of T3R1 parallel robot with uncoupled kinematic", Proc. of the 35<sup>th</sup> International Symposium on Robotics, Paris, France.
- Carbone, G., Ceccarelli, M. and Teolis, M., 2002, "A Numerical Evaluation of the Stiffness of CaHyMan (Cassino Hybrid Manipulator)", 2nd Workshop on Computational Kinematics CK 2001, Seoul, pp.145-154, 2001; Electronic

Journal of Computational Kinematics EJCK, http://www-sop.inria.fr/coprin/EJCK/EJCK.html, vol.1, n.1, paper n.14.

- Carbone, G., Lim, H.O., Takanishi, A. and Ceccarelli, M., 2003, "Optimum Design of a New Humanoid Leg by Using Stiffness Analysis", 12th International Workshop on Robotics in Alpe-Andria-Danube Region RAAD 2003, Cassino, paper 045RAAD03.
- Carvalho, J.C.M., Duarte, M.A.V., Ribeiro, C.R., 2001. Forward and Inverse Kinematic Model of a Parallel Cartesian Structure Using Neural Network. Proc. of Dynamic Problems of Mechanics (IX DINAME), pp: 207-211.
- Ceccarelli, M., Ottaviano, E. and Carbone, G., 2002, "A Study of Feasibility for a Novel Parallel-Serial Manipulator", Fuji International Journal of Robotics and Mechatronics, vol.14, n.3, pp.304-312.
- Clinton, C. M., Zhang, G. and Wavering, A. L., 1997, "Stiffness modeling of a Stewart-platform-based milling machine" Trans. of the North America Manufacturing Research Institution of SME, Vol. XXV, pp. 335-340, Lincon.
- Company, O., Pierrot, F. and Fauroux, J. C., 2005, "A method for modeling analytical stiffness of a lower mobility parallel manipulator", Proc. Of IEEE ICRA: Int. Conf. On Robotic and Automation, Barcelona, Spain.
- Corradini, C., Fauroux, J. C., Krut, S. and Company, O., 2004, "Evaluation of a 4 degree of freedom parallel manipulator stiffness", Proc. Of the 11<sup>th</sup> Word Cong. In Mechanism & Machine Science, IFTOMM'2004, Tianjin, China.
- Deblaise, D., Hernot, X. and Maurine, P., 2006, "A Systematic Analytical Method for PKM Stiffness Matrix Calculation"
- Dong, W., Du, Z. and Sun, L., 2005, "Stiffness influence atlases of a novel flexure hinge-based parallel mechanism with large workspace", Proc. of IEEE ICRA: Int. Conf. On Robotic and Automation, Barcelona, Spain.
- El-Khasawneh, B. S. and Ferreira, P. M., 1999, "Computation of stiffness and stiffness bounds for parallel link manipulator", Int. J. Machine Tools & manufacture, vol. 39(2), pp. 321-342.
- Gosselin, C. M., 1990, "Stiffness mapping for parallel manipulator", IEEE Trans. On Robotics and Automation, vol. 6, pp. 377-382.
- Huang, T., Zhao, X. and Whitehouse, D. J., 2002, "Stiffness estimation of a tripod-based parallel kinematic machine", IEEE Trans. On Robotics and Automation, Vol. 18(1).
- Imbert, J. F., 1979, "Anayse des structures par elements finis", Cepadues Editions.
- Li, Y. W., Wang, J. S. and Wang, L.P., 2002, "Stiffness analysis of a Steward platform-based parallel kinematic machine", Proc. of IEEE ICRA: int. Conf. On Robotics and Automation, Washington, US.
- Liu, X.-J., Jin, Z.-L. and Gao, F., 2000, "Optimum Design of 3-Dof Spherical Parallel Manipulators with Respect to the Conditioning and Stiffness Indices", Mechanism and Machine Theory, vol.35, n.9, pp.1257-1267.
- Majou, F., Gosselin, C. M., Wenger, P. and Chablat, D., 2004, "Parametric stiffness analysis of the orthoglide", Proc. Of the 35<sup>th</sup> International Symposium on Robotics, Paris, France.
- Martin, H. C., 1966, "Introduction to matrix methods of structural analysis", McGraw-Hill Book Company.
- Nof, S.Y. (Editor), 1985, "Handbook of Industrial Robotics", John Wiley & Sons, New York.
- Pai, D.K. and Leu, M.C., 1991, "Uncertainty and Compliance of Robot Manipulators with Application to Task Feasibility", International Journal of Robotics Research, vol.10, n.3, pp.200-213.
- Rivin, E.I., 1999, "Stiffness and Damping in Mechanical Design", Marcel Dekker Inc., New York.
- Simaan, N. and Shoham, M., 2002, "Stiffness Synthesis of a Variable Geometry Planar Robot", 8th International Symposium on Advances in Robot Kinematics ARK 2002, Lenarcic J. and Thomas F. (Editors), Kluwer Academic Publishers, Caldes de Malavella, pp.463-472.
- Tsai L.W., 1999, "Robot Analysis: The Mechanics of Serial and Parallel Manipulators", John Wiley & Sons, New York, pp.260-297.
- Yoon, W. K., Suehiro, T., Tsumaki, Y. and Uchiyama, M., 2004, "Stiffness Analysis and Design of a Compact Modified Delta Parallel Mechanism", Robotica, vol. 22, pp. 463-475.
- Zhang, D., 2000, "Kinetostatic analysis and optimization of paralle and hybrid architecture for machines tools", Ph.D Thesis, Laval University Quebec, Canada.
- Zhang, D., Xi, F., Mechefske, C. M. and Lang, S. Y. T, 2004, "Analysis of parallel kinematic machine with kinetostatic modeling method", Robotics and Computer-Integrated Manufacturing, vol. 20(2), pp. 151-165.

## 8. RESPONSIBILITY NOTICE

The authors are the only responsible for the printed material included in this paper.